



# Numerical Study on the Effects of Anti-Snow Deflector on the Wind-Snow Flow Underneath a High-Speed Train

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## ABSTRACT

In order to study the effect of the anti-snow deflector on the wind-snow flow underneath a high-speed train, Detached Eddy Simulation (DES) approach and discrete phase model (DPM) are used to simulate the wind-snow flow around the train. The distribution of snow particles underneath the train body is analyzed. Meanwhile, the influence of deflectors on the movement of snow particles around the train is investigated. The results show that lots of vortices shed from the bogie, and the entrainment vortices near the ground actuates the movement of the snow particles on the snow-covered track, which forms a wind-snow flow. The snow smoke around the train develops gradually from the bottom of the first bogie to the end of the tail car. The deflector installed in the front of the bogie will guide the vortices off the bogie region to the ground, which results in flying up more downstream snow particles and correspondingly the number of snow particles accumulated in the bottom of the rear car and around the rear skirt plate is increased. The installation position for the deflector has a certain effect on the snow accretion in the bogie region. When the deflector is installed in the front of the 2<sup>nd</sup> and 4<sup>th</sup> bogies, the snow particles captured in the bogie region are reduced by 42.3% and 15.6%, respectively.

**Keywords:** High-speed train; Bogie; Underbody flow; Wind-snow flow; Discrete phase model; Deflector; Anti-snow.

## NOMENCLATURE

$C_d$	particle resistance coefficient	$u_p$	particle velocity
$C_p$	pressure coefficient	$u_x$	time-averaged streamwise velocity
$d_p$	particle diameter	$u_y$	time-averaged lateral velocity
$g$	inertia acceleration	$u_z$	time-averaged vertical velocity
$H$	train height	$U$	normalized mean velocity
$L$	train length	$U_{ref}$	inlet velocity
$m_p$	particle mass	$W$	train width
$P$	static pressure	$\mu$	aerodynamic fluid viscosity
$P_\infty$	reference pressure	$\rho$	air density
$Re$	relative Reynolds number	$\rho_p$	particle density
$T_{ref}$	reference time	$\tau_r$	particle relaxation time
$u$	air velocity		

## 1. INTRODUCTION

When a high-speed train (HST) runs on the snow-covered track, the train-induced slipstream makes the covered snow flying off the track, causing the

snow accretion in the bogie region (Kamata and Yokokura, 2019). Due to the heat sources such as brake disc and traction motor in the bogie region (Milosevic *et al.* 2012; Petry *et al.* 2018; Nategh *et al.* 2019), some of the snow particles entering the

bogie region will melt into water droplets, and therefore the ice is easily formed. The freezing snow can damage the flexible components of the bogie and also significantly affect the movement of the elastic suspension, thus affecting the running safety of trains (Kloow, 2011). To minimize the impact of snow accretion on the running safety of trains, the railway manufactory has developed a variety of methods to remove the snow and ice on the bogie.

The process of snow particles being carried into the bogie region by the slipstream is a phenomenon of aeolian transport. The method of reducing snow particles entering the bogie region can be studied in terms of flow control which slows down the snow accumulation speed in the bogie region. For instance, Shishido *et al.* (2009) investigated the influence of the side deflector of the train skirt plate on the snow accretion in the bogie region in a snowfall wind tunnel. studied the influence of the triangular prism spoiler on the snow accretion in the bogie region in a low-temperature wind tunnel. Gao *et al.* (2019) used discrete phase model (DPM) to study the effects of the deflector at the bottom of the end wall of the bogie cavity on reducing the snow accretion in the bogie region. The effects of deflector, shape of bogie cavity, anti-snow shields on reducing snow accretion of bogie and brake clamp were investigated using DPM (Wang *et al.* 2018a,b, 2019a,b). The previous studies show that installing a deflector under or on the side of the train or increasing the inclination of the end walls of the bogie cavity can effectively reduce the snow accumulation in the bogie region.

With the increasing availability and accuracy, computational fluid dynamics (CFD) is widely used in the prediction of snowdrift around buildings (Tominaga, 2018). The approaches include Euler-Euler and Euler-Lagrangian methods. Discrete phase model (DPM) belongs to Euler-Lagrangian frames, which has been proved to be highly accurate in solving the gas-solid two-phase flow problems (Patankar and Joseph, 2001; Chiesa *et al.* 2005; Pankajakshan *et al.* 2011). Paradot *et al.* (2014) verified the snow accretion on a bogie calculated using DPM with the experimental data. And the snow accumulation process on the bogie was simulated using DPM together with mesh deformation tools. Gao *et al.* (2019) investigated the snow accretion on the bogie of a HST with DPM and tested the snow accretion on the bogie with sawdust substituting snow particles. The distribution of snow particles on the surface of bogie calculated by DPM was consistent to the experimental results. As the DPM contains all the information of particle motion, it is widely applied to study the snow accretion of bogies (Wang *et al.* 2018a,b, 2019a,b; Gao *et al.* 2019; Xie *et al.* 2017).

CFD has been also widely used in the research of the flow around trains or underbody flow of trains. Li *et al.* (2019) compared the different RANS turbulence model on the flow around a simplified train. It is found that the shear stress transport (SST) model has an excellent ability in the prediction of aerodynamic forces of the train. Soper *et al.* (2018)

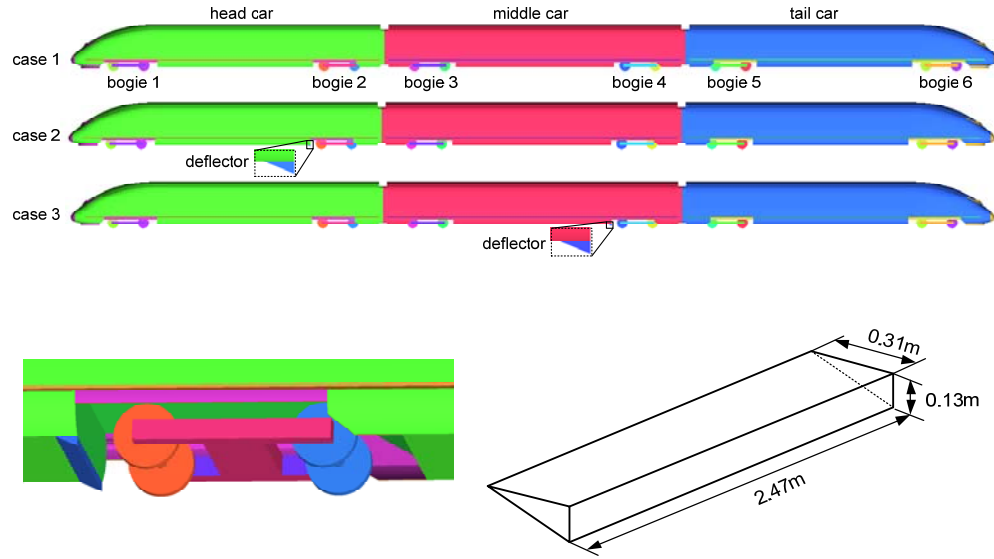
compared the performance of CFD and dynamic model wind tunnel test method for the flow beneath the train. Zhu and Hu (2017) studied the force characteristics of airflow beneath a HST bogie using Detached Eddy Simulation (DES) approach. Paz *et al.* (2017,2019) used delayed DES (DDES) and dynamic mesh to investigate the effects of true shape track beds and ballasts on underbody flow as trains pass. DES approach has high accuracy in predicting the separation position and aerodynamics of the large separation flow of ground cars (Kapadia *et al.* 2003; Sreenivas *et al.* 2006; Guilmineau *et al.* 2011; Kang *et al.* 2012; Li *et al.* 2020). It is proved that DES approach has the ability to accurately predict in the slipstream, wake and underbody flow of HST (Zhang *et al.* 2016; Wang *et al.* 2017,2018; Xia *et al.* 2017). Meanwhile, it also performs well in solving flows around aircraft landing gear (Hedges *et al.* 2002, Xiao *et al.* 2013, Dong *et al.* 2018) and bogie (Sima *et al.* 2008; Wang *et al.* 2019a). DES approach has a strong dependence on the computational mesh. The inappropriate mesh will start the sub-mesh model in advance, which will lead to the stress loss of boundary layer and mesh-induced separation (Vatsa and Lockard, 2010). The Improved Delayed DES (IDDES) model reduces the dependence on mesh and makes DES class approach more suitable for engineering field. In this paper, IDDES and DPM are selected to study the influence of anti-snow deflector on wind-snow flow beneath HST.

This paper mainly studies the influence of the anti-snow deflector on wind-snow flow underneath the train. The content of this paper is as follows. In section 2, the train model, computational domain and boundary conditions, snow injector settings and calculation mesh are described in section 2. And in the next section, the characteristics of snow around the train and the influence of anti-snow deflector of bogie on snow particle transport beneath the train are analyzed and discussed. Finally, the conclusion is described in section 4.

## 2. NUMERICAL SETUPS

### 2.1 Geometric Model

The numerical model consists of 3 cars of CRH3 high-speed train, including one head car, one intermediate car, one tail car and six simple bogies. For simplicity, the 1<sup>st</sup> bogie is named as bogie 1, and the same rules for the remaining bogies. The length, width and height of the train are  $L=76.5$  m,  $W=3.3$  m and  $H=3.9$  m, respectively. Three different models (Case 1, Case 2 and Case 3) are established to analyze the influence of anti-snow deflector on the wind-snow flow. Case 1 does not contain any deflector. Case 2 installed a deflector in the front of the 2<sup>nd</sup> bogie, and Case 3 in the front of the 4<sup>th</sup> bogie. The installation position and size of the deflector are shown in Fig. 1. To compare with the numerical results of the vertical arrangement of the snow ejector in front of the car, the shape and installation position of the deflector in this study is the same as the one used by Gao *et al.* (2019).



**Fig. 1. Train model and anti-snow deflector.**

All simulations presented are completed in the commercial code Fluent 15.0. The computational domain is shown in Fig. 2. Its length, width and height are 216.5 m, 60 m and 100 m, respectively. The inlet is 40 m away from the nose tip of the head car. The outlet is 100 m away from the nose tip of the tail car. The bottom of the train is 0.35 m away from the ground. In the numerical simulation, the inlet is set as velocity inlet. The outlet is set as a pressure outlet. The ground is set as a sliding wall. The side and top surfaces are set as symmetrical wall. The wheels are set as rotating wall. The specific boundary conditions are shown in Table 1.

To investigate the characteristics of the snow particles on the walls, the bogie and the car body use a user-defined function (UDF) in discrete phase boundary conditions. The UDF is used to count the total number of impact snow particles on the wall. The statistical results are stored in the user-defined memory (UDM). A predefined macros DEFINE\_DPM\_BC provided by Fluent is used to define the UDF to specify user-defined boundary conditions for particles. The function is executed every time a particle touches a boundary of the domain. When the particle hits the wall, the value of the UDM that stores the number of particle impacts will be increased by 1, and then the state of the particle will be marked as PATH\_END. For the return status PATH\_END, the particle will be assumed to escape from the domain.

To save computing resources, a transverse sampling plane is set at the rear of the train in the computational domain (see Fig. 2). With the custom sampling function (UDF), snow particles passing the sampling surface will be released from the computational domain and will not be considered in the subsequent calculation. Therefore, the number of tracking snow particles can be decreased substantially. In all, the number of particles tracked

in this study is about 26 million for a stable wind-snow flow.

## 2.2 Methodology for CFD Analysis

In this paper, an IDDES approach (Shur *et al.* 2008) based on the SST  $k-\omega$  model is used to capture more vortices in the wake flow between the train and track bed. As a RANS-LES hybrid technique, IDDES approach combines the advantages of the delayed eddy simulation (DDES) and the wall-modelled large eddy simulation (WMLES). The DDES approach provides the delayed option to protect the boundary layer from the limiter. The complete formulations for the IDDES can be referred by Shur *et al.*(2008) and Gritskevich *et al.*(2012)

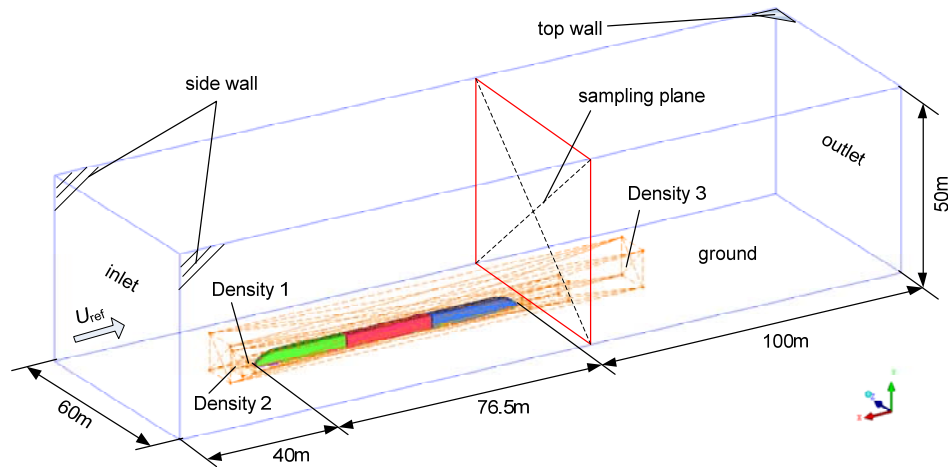
The trajectory of snow particles predict by the DPM. The snow particles are assumed to be spheres without considering the impact of the collision between them. The force balance equation of snow particles is as follows:

$$m_p \frac{du_p}{dt} = m_p \frac{u - u_p}{\tau_r} + m_p \frac{g(\rho_p - \rho)}{\rho_p} \quad (1)$$

Where  $m_p$  is the particle mass;  $u$  is air velocity;  $u_p$  is snow particle velocity;  $\rho$  is air density, and  $\rho = 1.225 \text{ kg/m}^3$ ;  $\rho_p$  is snow particle density;  $\tau_r$  is particle relaxation time calculated by:

$$\tau_r = \frac{\rho_p d_p^2}{18\mu C_d Re} \quad (2)$$

Here,  $\mu$  is aerodynamic fluid viscosity,  $d_p$  is the diameter of snow particle,  $C_d$  is the resistance coefficient of particle,  $Re$  is the Reynolds number,



**Fig. 2. Computational domain.**

which is defined as

$$Re \equiv \frac{\rho d_p |\vec{u}_p - \vec{u}|}{\mu} \quad (3)$$

In the previous simulations of the wind-snow (Wang *et al.* 2018a,b, 2019a,b; Gao *et al.* 2019), snow particles injected into the train underbody along the flow direction were arranged by a transverse jet plane. In this way, the snow particles cover the entire gap between the bottom of car and the rail. The snow particles injected in the front of the car gradually reach the downstream bogies region over time. Due to the capture of the upstream walls of the train, the number of snow particles entering the downstream region beneath the car will decrease progressively. This method ignores the vertical development of wind-snow flow beneath HST and is not suitable for the underdeveloped situation. Observing the distribution of wind-snow flow caused by the running train in actual (Kloow, 2011), we can see that the snow smoke develops from the region beneath of the head car. Its coverage becomes higher gradually, and finally forms the snow smoke that surrounds the whole train.

Different from the layout of the snow injector in the literature (Wang *et al.* 2018a,b, 2019a,b; Gao *et al.* 2019), we adopts the horizontal layout of the snow injector on the ground beneath the train. The particles are emitted downstream in the horizontal direction. The arrangement of the injector is shown in Fig. 3. The horizontal inject plane is 5 cm above the ground, the injection points are arranged at an interval of 0.1m (760×40=30400 points). The train speed is 70 m/s. At the injection point, a particle with an initial velocity (-70, 0, 0) m/s was launched in the streamwise direction every 0.0005 s, the diameter is 0.2 mm and density is 150 kg/m<sup>3</sup>. The mass flow rate of the snow particles emitted from the injector was 0.0382 kg/s.

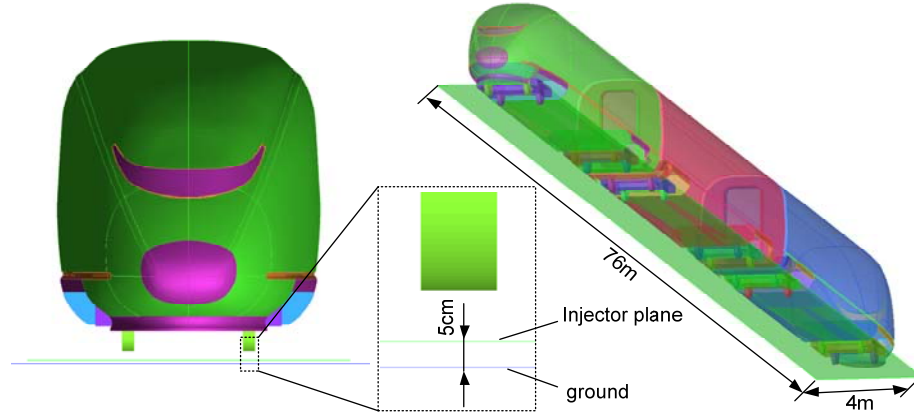
The stochastic tracking approach was adopted for DPM. The random walk model (RWM) (Gosman

and Loannides, 1983) was selected to simulate the influence of turbulence on the diffusion of snow particles. In the stochastic tracking approach, it predicts the turbulent dispersion of particles by integrating the trajectory equations for individual particles using the instantaneous fluid velocity,  $\vec{u} + u'(t)$ , along the particle path during the integration. Fluent uses a stochastic method to determine the instantaneous gas velocity. Firstly, the steady flow field is solved, and then the transient calculation is started with the steady flow field. The DPM calculation is started when the transient flow time reaches 1.0 s. The time step of  $\Delta t_{IDDES} \approx 0.0025T_{ref}$  ( $T_{ref} = H / U_{ref}$ ) is required due to the restrictions of the Courant number (Wang, *et al.* 2017). In this paper, the time step for each case is fixed with  $1 \times 10^{-4}$  s, and the CFL number of most cells is less than 1. Each time step iterates 20 times, and the total solution time of DPM is 2.5 s.

**Table 1 Boundary conditions**

boundary	boundary condition type	parameter
inlet	Velocity inlet	X=-70 m/s, Escape Turbulence Intensity:1% Turbulence Viscosity Ratio: 10%
outlet	Pressure outlet	0 Pa, Escape
side and top wall	Symmetry	-
ground	Moving wall: transition	X=-70 m/s, Escape
wheel	Moving wall: rotation	152 rad/s, UDF
bogie	No-slip wall	UDF
train body	No-slip wall	UDF

The finite volume method is used to discretize the governing equations. The equations (the instantaneous continuity and momentum equations,



**Fig. 3. Location and size of snow injector plane below the train.**

The turbulence kinetic energy,  $k$ , and the specific dissipation rate,  $\omega$ , transport equations) is solved by SIMPLE algorithm. Bounded central differencing scheme is adopted for the momentum term. The pressure terms, turbulent flow energy and turbulent diffusivity is discrete using second order upwind scheme. And the time term is discrete using bounded second order implicit scheme.

### 2.3 Computational Mesh and Validation

Two sets of tetrahedral grids named as medium and fine mesh are generated. The medium and fine meshes have 57 and 98 million volume cells, respectively. Figure 5 shows the comparison in the pressure coefficient  $C_p$  for numerical and the experimental results (Xia *et al.* 2017) at the centerline of the upper surface of the head and tail cars. The static pressure on the train and mean velocity component in the travelling direction are normalized as follows:

$$C_p = (P - P_\infty) / (0.5\rho U_{ref}^2) \quad (3)$$

$$U = \sqrt{(U_{ref} - u_x)^2 + u_y^2 + u_z^2} / U_{ref} \quad (4)$$

Where  $P$  is the static pressure,  $P_\infty$  is the reference pressure,  $U_{ref}$  is the inlet velocity,  $u_x$ ,  $u_y$  and  $u_z$  are the streamwise, the lateral and the vertical components of air velocity.

The time-averaged pressure coefficients  $C_p$  obtained using two different grids match the experimental values (Xia *et al.* 2017). Therefore, the medium mesh is chosen to simulate the wind-snow flow around the train. The details of the medium mesh in this study are shown in Fig. 4. Two mesh refinement regions are added around the train, and a local mesh refinement region is set at the bottom of the train. The near-wall region is divided into 15 layers of prism mesh. The thickness of the first layer is 0.5 mm, which is the same as that in the literature (Paz *et al.* 2019). In this paper,  $y^+$  is mainly in the range of 5 and 60. IDDES uses

LES method in the vortex core region far away from the wall. If all scales eddies are to be calculated, the existing computing capacity cannot meet the requirements of mesh quantity. Therefore, only large-scale eddies played a significant role in aerodynamic forces and particle diffusion are considered in this paper by using the medium mesh. The maximum mesh size of the region beneath the train for the medium mesh is 32 mm.

Figure 6 shows the comparison in the normalized mean velocity component  $U$  using the numerical simulation, the full-scale test (Jönsson *et al.* 2014) and the scale model test (Deeg *et al.* 2008). Due to the difference in the length of the adopted train, the horizontal coordinate data is aligned for comparison. The full scale measurements was conducted for an Italian high-speed train ETR500 (Jönsson *et al.* 2014), and the scale model test was measured on the ICE3 (Deeg *et al.* 2008). The numerical results accurately capture the peak velocity fluctuation when the head and tail cars pass by. The velocity around the head and tail cars close to the test results, and the velocity variation trend matches the measured data for ETR500. However, it should be noted that the underbody flow is quite sensitive to the geometry (Deeg *et al.* 2008; Jönsson *et al.* 2014; Wang *et al.* 2018; Zhang *et al.* 2018). The train model adopted in this paper is not exactly the same as that in the experiments. Therefore, the calculated result is different from the experimental data in certain. The comparison between the calculated results and the experimental results shows that the numerical method and the medium mesh used in this paper are reasonable.

## 3. RESULTS AND DISCUSSION

### 3.1 Snow Smoke Around the Train

Figure 7 shows the distribution of snow particles around the train at different DPM simulation time  $t_p$ . The snow particles are colored by the dimensionless time  $t_{pr}/t_p$  ( $t_{pr}$  is the particle residence time). At first, the snow particles on the ground

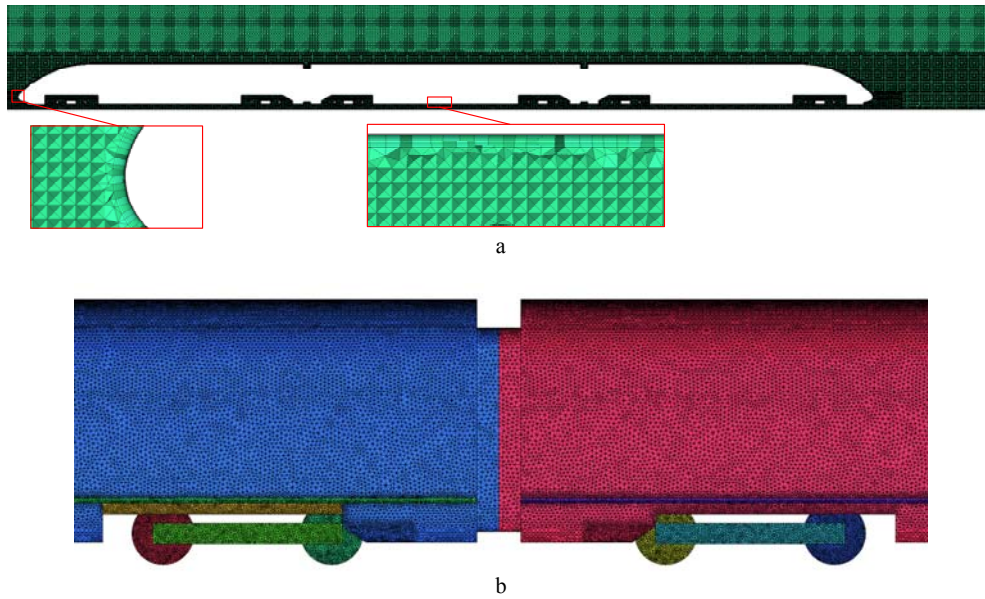


Fig. 4. Computational mesh: (a) volume mesh; (b) surface mesh.

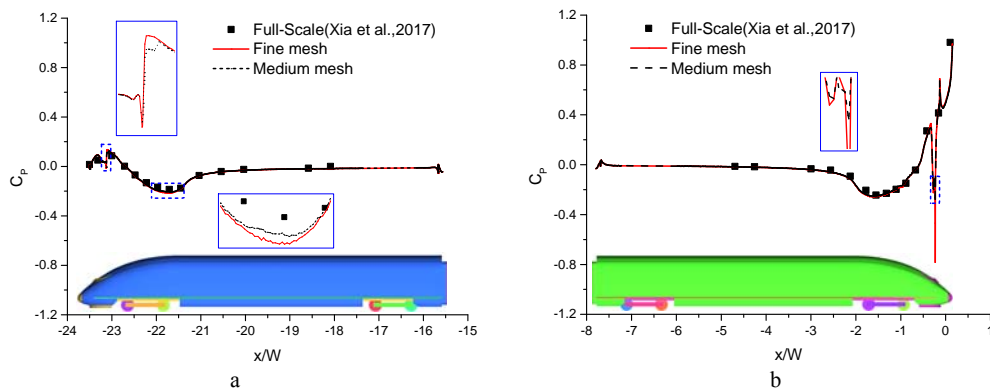
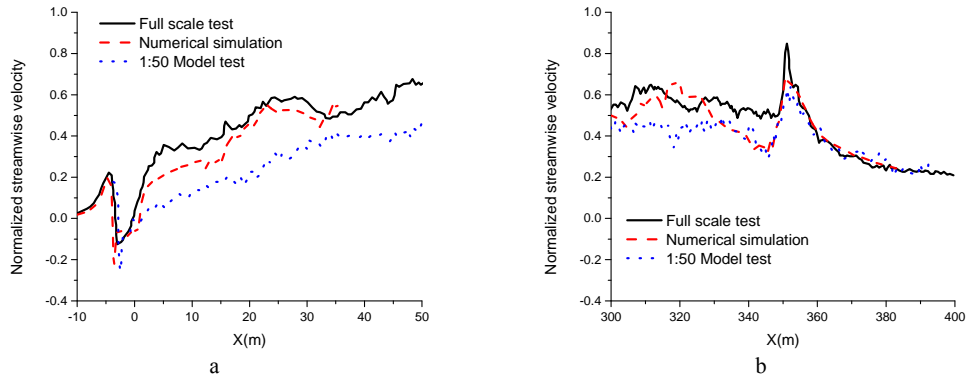


Fig. 5. Comparison in pressure coefficients at the center line of the upper surface of the car body between numerical simulation and experiments: (a) tail car; (b) head car.

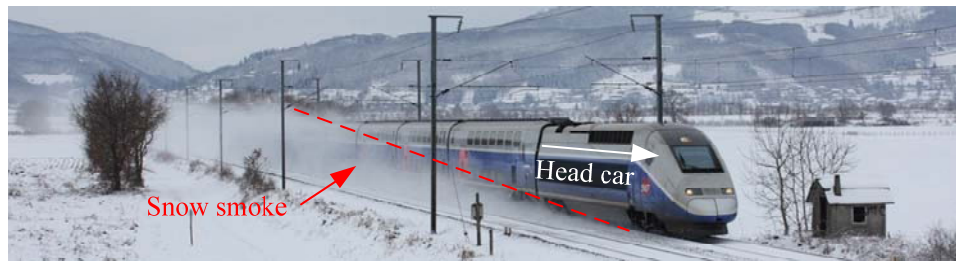
gradually move to the upper region around the train. After 0.1 s, the particles reach the bottom surface of the train, and 0.5 s later, the particles reach the height of the train. The particle residence time  $t_{pr}$  of most snow particles around the train is less than  $t_p$ . It can be seen that the wind-snow flow around the train has reached the relatively stable state after  $t_p = 1.5$  s. At that state, the distribution height of snow particles around the train gradually increases from the head car to the tail car, reaching the same height of the train at the tail car. In particular, as the snow particles rolled up by the underbody airflow are too late to enter the bogie 1 region, the snow particles are less accumulated in the region of the bogie 1. The distribution trend of snow particles along the longitudinal direction of the train using numerical simulation is consistent with the actual

situation.

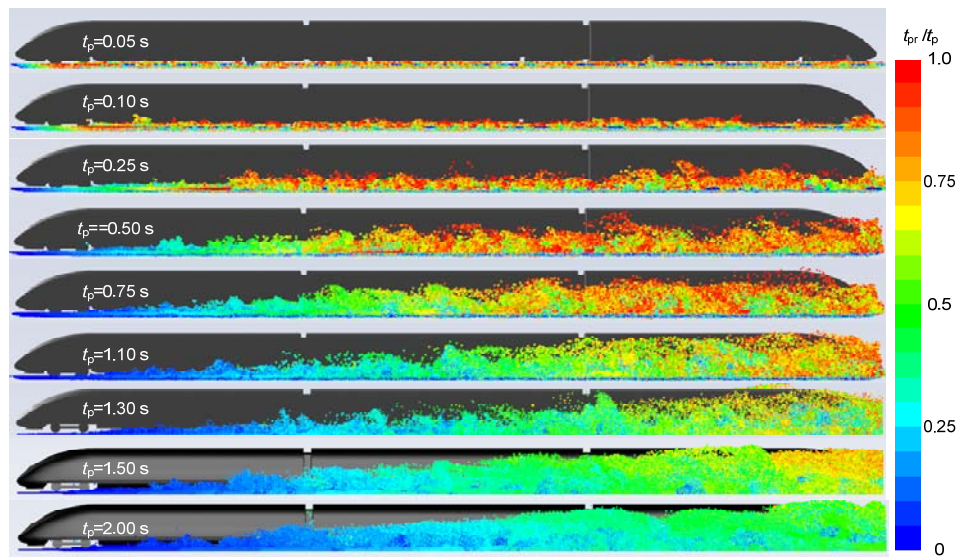
In order to compare the distribution of snow concentration along the longitudinal direction beneath the train, three longitudinal monitoring lines parallel to the X axis are set up and shown in Fig. 8. The lines 1 and 2 are located beneath the car and line 3 is located on the side of the skirt plate. The average volume fraction of snow particles on the monitoring lines are calculated (see Fig. 9). It shows that the snow particle concentration under the train (line 1 and 2) begins to increase rapidly at the rear of the bogie 1 and reaches the average level at the middle of the head car. The peak value of snow concentration underneath the train appears below the bogie, and the snow concentration below the bogie 3 is the highest. The snow concentration on the skirt plate side (line 3) increase gradually from the rear of the bogie 1 and reached the average level at the central of the middle car. It shows that



**Fig. 6.** Comparison in the normalized mean velocity component at the height of top rail obtained using full-scale, model scale tests and numerical simulation: (a) head; (b) tail.



a



b

**Fig. 7.** Snow particles distribution around the train: (a) snow smoke around train (Paradot *et al.* 2014); (b) snow particles distribution around the train at different times.

the snow concentration on the side of the skirt plate is about twice as small as that underneath the car.

### 3.2 Effects of Anti-Snow Deflector on Snow Accretion of Bogie

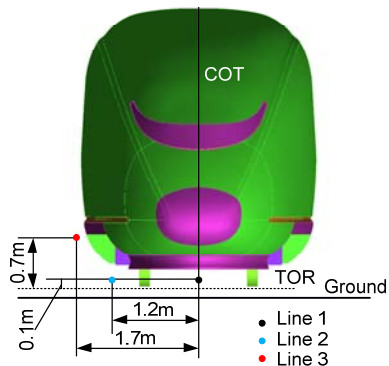
Table 2 shows the change rates of snow particles

captured by each car body and bogies for different cases during  $t_p=1.0\sim 2.0$  s. From the comparison between Case 2 and Case 1, it can be seen that when the deflector is installed at the bogie 2, the snow particles captured by the bogie 2 are reduced by about 59.6% for Case 2. However, the snow

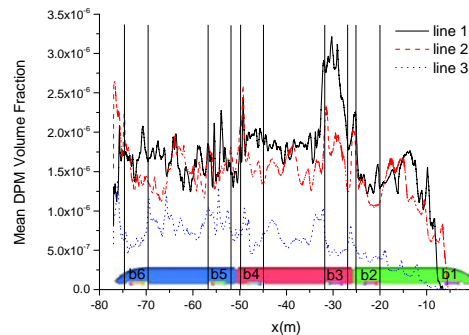
**Table 2 Change rates of snow particles captured on each part in different case within 1.0 s to 2.0 s**

Parts	Total number of captured snow particles			Change rates	
	Case 1	Case 2	Case 3	Case 2	Case 3
Bogie 1	1665	2367	1014	0.422	-0.391
Bogie 1 cavity	2341	2167	1774	-0.074	-0.242
Bogie 2	754988	304474	770589	-0.597	0.021
Bogie 2 cavity	252036	276774	283001	0.098	0.123
Bogie 3	1194021	721065	1454058	-0.396	0.218
Bogie 3 cavity	461380	480326	559440	0.041	0.213
Bogie 4	1382581	987232	838483	-0.286	-0.394
Bogie 4 cavity	548800	424002	679404	-0.227	0.238
Bogie 5	1443515	1125120	1216863	-0.221	-0.157
Bogie 5 cavity	724119	484889	691298	-0.330	-0.045
Bogie-6	1469200	923798	1072566	-0.371	-0.270
Bogie 6 cavity	695122	411812	501919	-0.408	-0.278
Carbody head	594328	727594	616137	0.224	0.037
Carbody mid	2382022	3083932	3185833	0.295	0.337
Carbody tail	3202513	2370367	3592394	-0.260	0.122
Deflector	-	340514	486162	-	-

particles captured on the inner surface of the bogie 2 cavity are increased by 9.8%. The total number of snow particles captured on the walls of the entire region of bogie 2 (bogie 2 and bogie 2 cavity) for Case 2 is reduced by 42.3%. Meanwhile, due to the impact of the upstream deflector installed in the front end of the bogie 2, the snow particles captured by bogies 3, 4, 5 and 6 are reduced by 39%, 28%, 22% and 37%, respectively. Moreover, it results in the number of snow particles captured by the head and middle car increased by 22% and 29%, respectively. Although the deflector at the bogie 2 can reduce the number of snow particles captured on the walls which is adjacent to the bogie 2. It will disturb more snow particles from ground, resulting in more snow particles captured by the downstream car body.



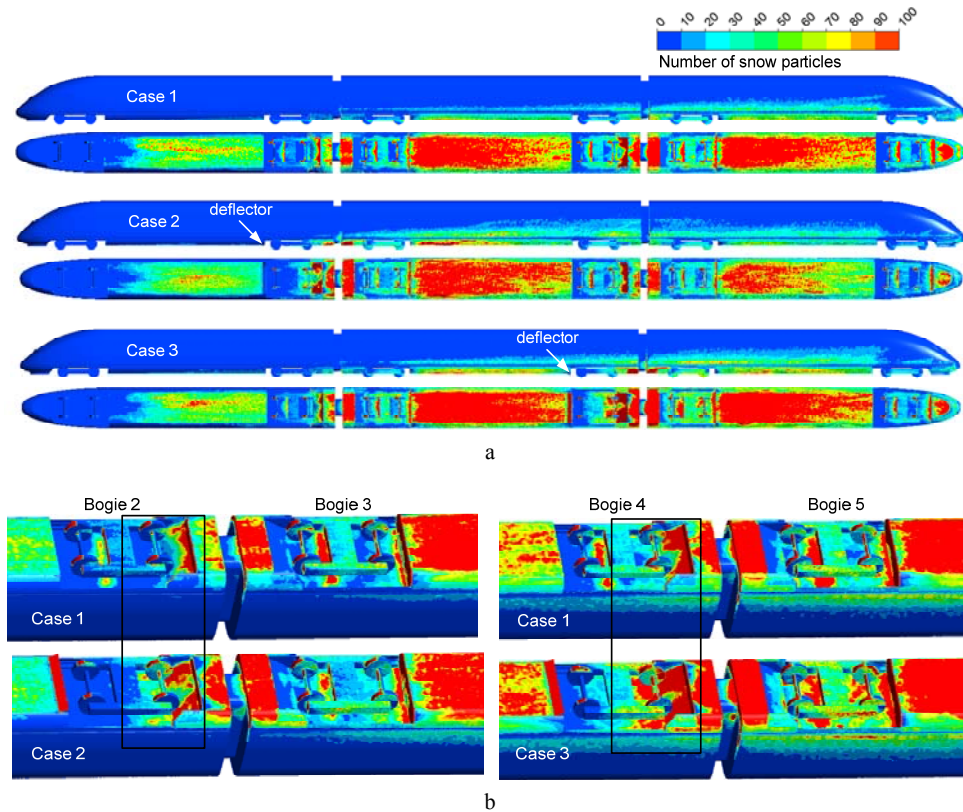
**Fig. 8. Locations of monitoring lines of snow particle concentration.**



**Fig. 9. Mean volume fraction of snow particle at the monitoring line for Case 1.**

From the comparison between Case 3 and Case 1, it shows that when the deflector is installed at the bogie 4, the snow particles captured by the bogie 4 are reduced by about 39.4%. However, the snow particles captured on the inner surface of the bogie 4 cavity are increased by about 23.8%. Although the deflector can significantly reduce the number of snow particles hitting the bogie 4, it will also significantly increase the number of snow particles hitting the walls of the bogie 4 cavity. The amount of snow particles captured on the bogie 4 region (bogie 4 and bogie 4 cavity) is reduced by only about 15.6%. In addition, due to the impact of the deflectors at the bogie 4, the number of snow particles captured by the middle and the tail cars increase by 33.7% and 12.2%, respectively. This features show that the deflector at the bogie 4 can reduce the number of snow particles captured on the walls adjacent to the bogie 4. Meanwhile, it will also disturb more snow particles, resulting in more





**Fig. 10.** The number of snow particles captured on the wall within 0 to 2.0 s: (a) the total number of snow particles captured on the wall within 0 to 2.0 s; (b) comparison in the total number of snow particles captured on the wall of the bogie region among Case 1, Case 2 and Case 3.

snow particles captured by the downstream car body.

Figure 10 presents the distribution of the number of snow particles captured by the train body and the bogie walls during  $t_p = 2.0$  s. By comparing Case 2 with Case 1, it can be seen that the deflector effectively reduces the number of snow particles captured in the front part of the bogie 2. However, it also increases the number of snow particles captured in the rear part of the bogie 2. At the same time, the deflector at the bogie 2 will increase the number of snow particles captured on the side of the middle car. And the height of the capture area of the snow particle is also higher. It is noted that the number of snow particles captured on the region of the bogie 6 and the bottom region of the tail car is reduced. The reason may be that the deflector of the head car causes the middle car to capture more snow particles, resulting in lesser snow particles being captured by the tail car.

With comparison to Case 1, the deflector for Case 3 can effectively reduce the number of snow particles captured in the front part of the bogie 4. But also significantly increase the number of snow particles captured by the bogie 4 cavity and car body at the rear of the bogie 4. In addition, the deflector at the bogie 4 of middle car will increase the number of snow particles captured on the side of the

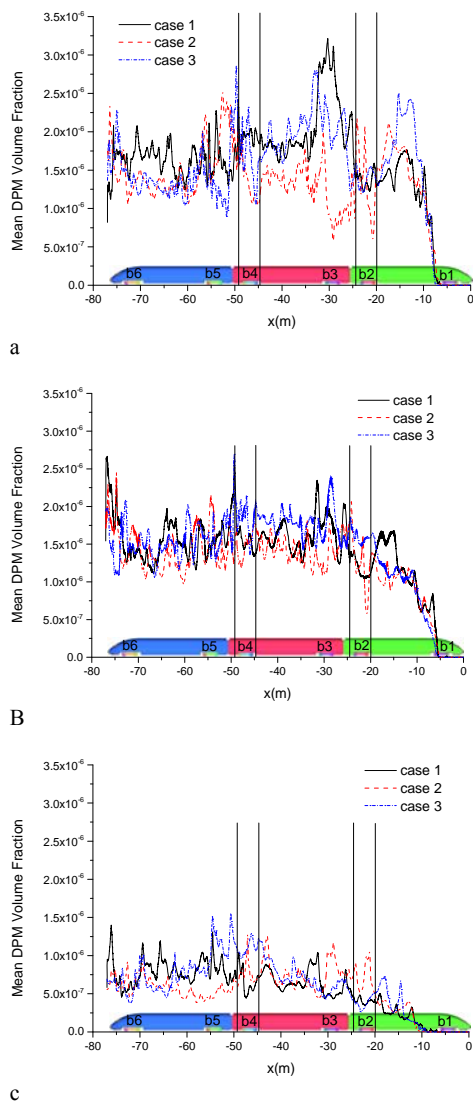
downstream tail car, and the height of the capture area of the snow particle is also higher.

In general, the deflector can significantly reduce the number of snow particles hitting the wall of the front part of the bogie. It slightly increases the number of snow particles hitting the wall of the rear region of the bogie. When the deflector is installed at the bogie 2 or 4, the snow particles captured in the bogie region are reduced by 42.3% and 15.6%, respectively. The effect of preventing snow in the bogie region is different with the deflector in a different bogie position. The anti-snow performance of the deflector at the front of the train is better than that in the rear.

Figure 11 shows the time-averaged volume fraction of snow particles along the longitudinal monitoring lines for the three cases. With comparison to Case 1, it can be seen that the snow particle concentration at the bottom of bogie 2 (lines 1 and 2) decreases at first and then increase along the flow direction for Case 2. At the same time, the deflector can obviously reduce the snow concentration below the downstream middle car. Moreover, the influence range of the deflector extends to the whole underbody of middle car. The impact of the deflector of the bogie 2 on the snow concentration in the edge beneath the bogie (line 2) is less than that in line 1. Comparing Case 3 with Case 1, it

shows that the impact of the deflector of the bogie 4 on the snow particle concentration on downstream car is similar to that of the deflector at the bogie 2. However, the concentration variation downstream deflector is smaller than that for Case 2.

With comparison to Case 1, due to the influence of the deflector of the bogie for both Cases 2 and 3, the snow particle concentration on the side of skirt plate (line 3) of the bogie 2 or 4 is significantly increased. Furthermore, the snow concentration on the side of the downstream skirt plate of the bogie 3 and 5 decreases at first and then increases. In Case 2, the snow particle concentration on the side of the tail car skirt plate slightly decreased. Since the position of line 3 is higher than lines 1 and 2, the snow particle concentration at line 3 is much lower than that of lines 1 and 2.



**Fig.11. Time-averaged volume fraction of snow particles along longitudinal monitoring line within 2.0 s to 2.5 s: (a) line 1; (b) line 2; (c) line 3.**

In general, the deflector can reduce the snow particle concentration beneath the car, but increase the snow particle concentration on the downstream skirt plate side. These areas may be more susceptible to the direct impact by the deflector and are more prone to accumulate snow.

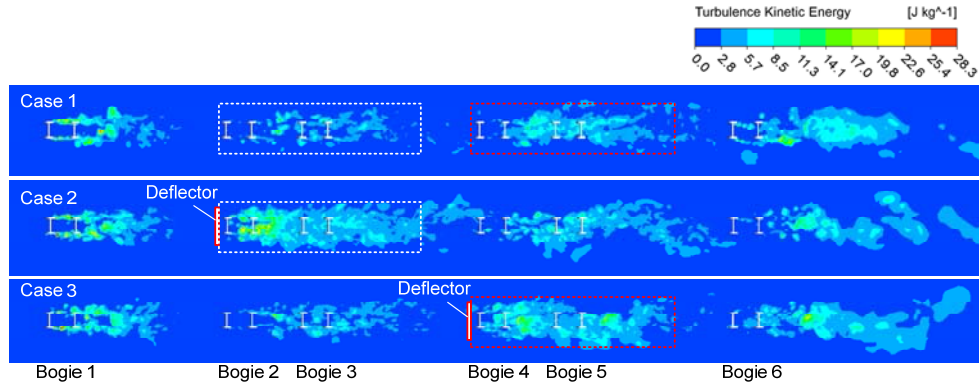
Figure 12 shows the turbulent kinetic energy distribution at the ejection surface beneath the train for the three cases. For Case 2, the turbulent kinetic energy in the lower region of the bogie 2 and 3 was larger than that for Case 1. For Case 3, the turbulent kinetic energy in the lower region of the bogie 4 and 5 was also larger than that for Case 1. Due to the effects of the deflector installed in front of the bogies 2 or 4 cavities, the turbulent kinetic energy in the lower region of the bogie increases obviously. The high turbulence intensity helps the airflow to blow more snow particles on the ground and increases the amount of snow particles carried by the downstream airflow.

To further explain the impact of the deflector on snow transportation underneath the train, the vortex cores around the bogie for the three cases are shown in Fig. 13. It is post-processed using commercial software EnSight 2019R1. The vortex cores help visualize the centers of swirling flow. Comparing with Case 1, more vortex cores are close to the ground below the bogie 2 or 4 downstream the deflector for Cases 2 or 3. The vortex shedding at the edge of the bogie and the deflector will be guided toward to the ground, and the time and position of the downwash vortices reaching the ground will be advanced compared to the original models. Furthermore, the downstream vortices behind the deflector will suck up more snow particles on the ground, thereby causing the amount of snow particles captured on the downstream wall increased.

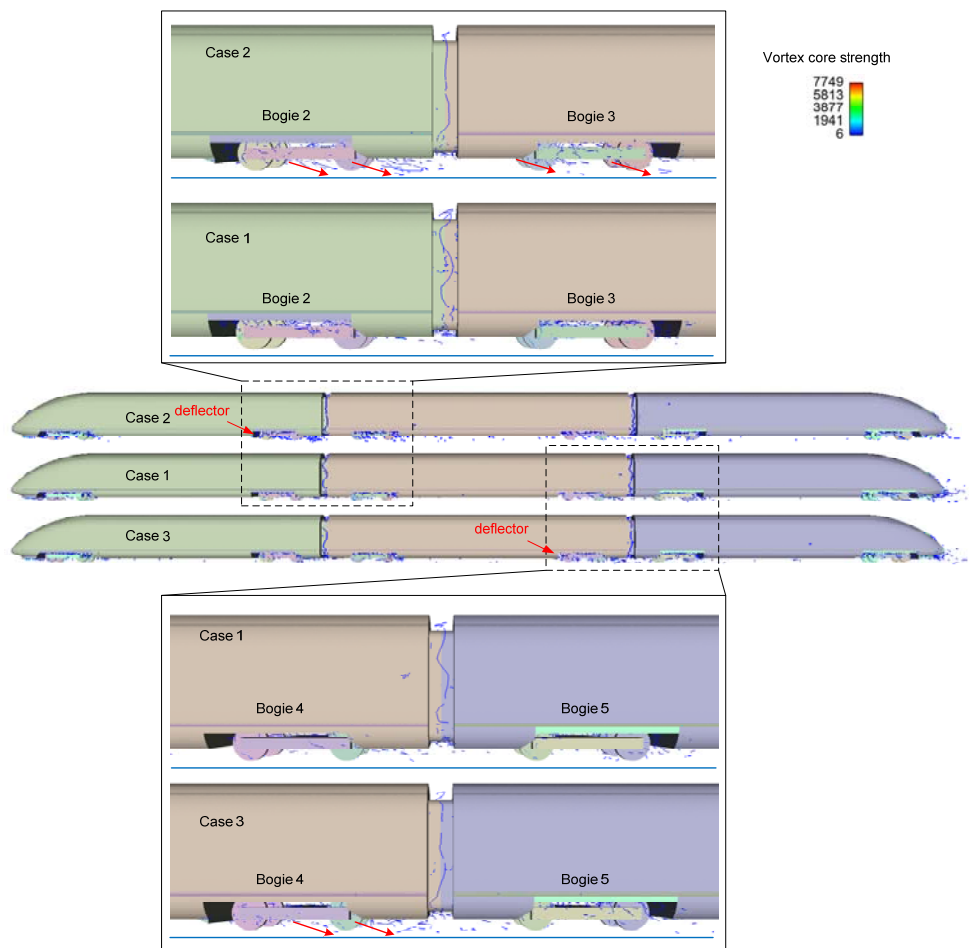
#### 4. CONCLUSION

The influence of the anti-snow deflector on the wind-snow flow underneath the high-speed train is investigated using the numerical simulation combining IDDES approach and DPM. Based on the results of numerical simulation, the following conclusions can be drawn.

- (1) The wind-snow flow around the train gradually develops from the head car to the tail car, increases gradually backward, and reaches the equilibrium state at the tail car. The concentration of snow particles beneath the train increases gradually from the head car, and gradually reaches a stable level at the central of the middle car.
- (2) The deflector can significantly reduce the number of snow particles hitting the wall of the front region of the bogie. Meanwhile, the deflector can also disturb more snow particles from ground, resulting in more snow particles captured by the wall of the rear region of the bogie and the downstream car body. In addition, the deflector leads to an increase in the snow particle concentration on the side of the downstream skirt plate.



**Fig. 12.** Turbulent kinetic energy in the ejection surface at  $t_p = 2.0$  s.



**Fig. 13.** Vortex core lines around the bogie at  $t_p = 2.0$  s.

- (3) The deflector guides the vortex shedding from the bogie to the ground, which increases the intensity of turbulent kinetic energy beneath the bogie. Furthermore, the downstream vortices behind of the deflector suck up more snow particles on the ground, which increase the number of snow particles carried by the downstream airflow and captured on the downstream surface of the train.
- (4) The installation position for the deflector has a certain effect on the snow accretion in the bogie region. When the deflector is installed at the bogie 2 or 4, the snow particles captured in the bogie region are reduced by 42.3% or 15.6%, respectively.

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